

[54] PROGRAMMABLE BREAKAWAY CLUTCH SYSTEM WITH COLLAPSIBLE FAILURE MODE

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[58] Field of Search 192/56 R, 56 F, 150; 901/29, 46, 49; 464/36, 38, 39

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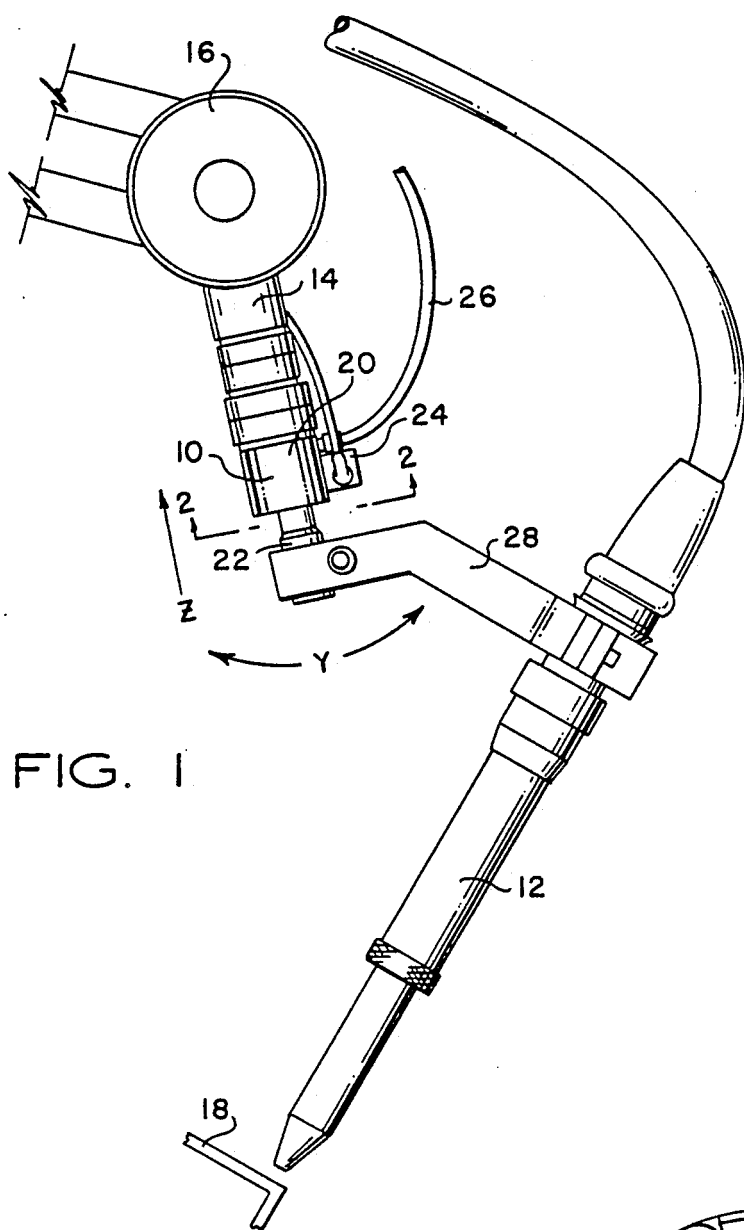


FIG. 1

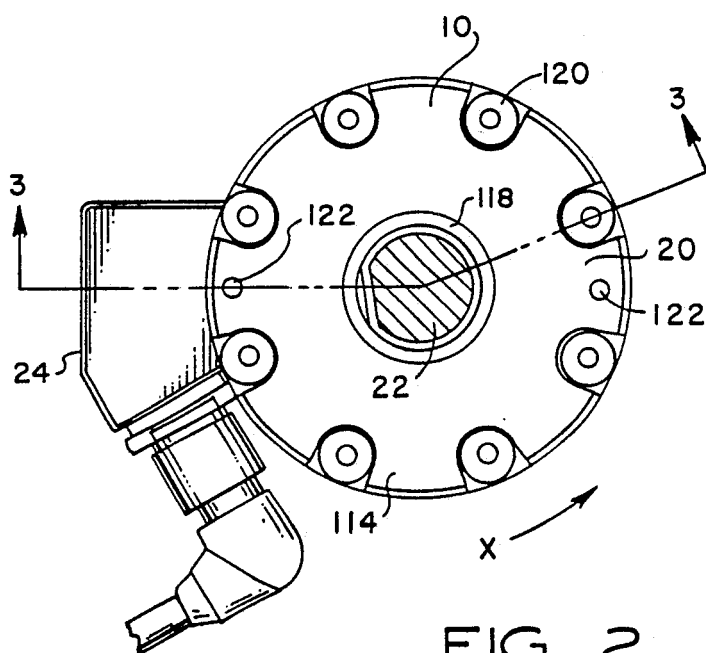
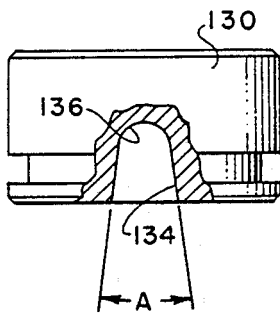
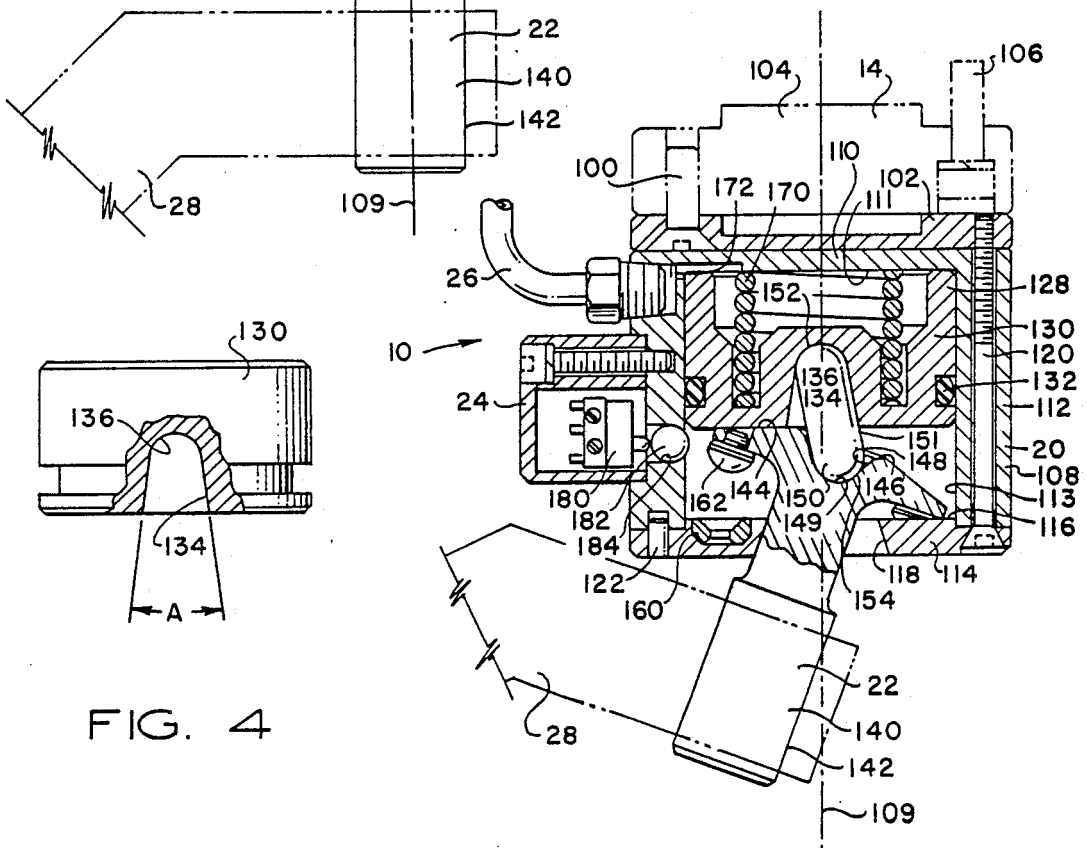
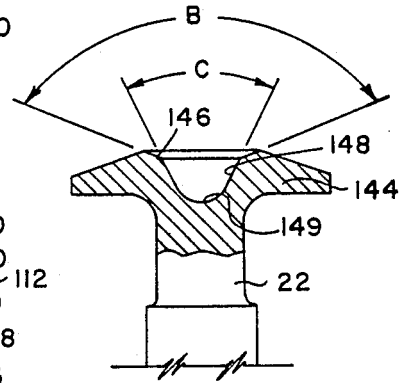
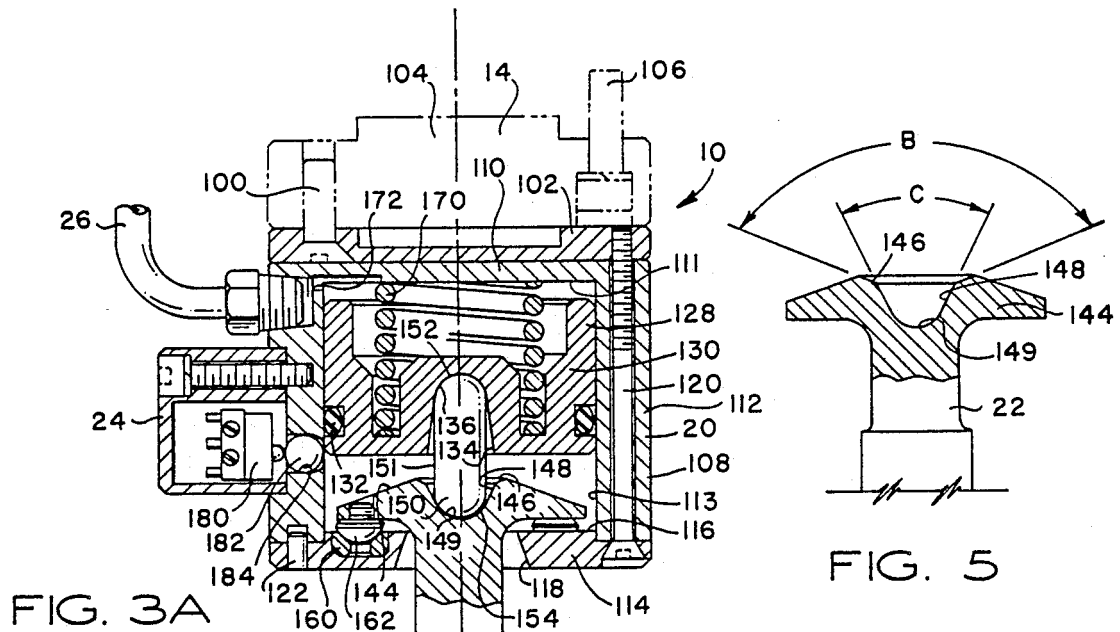


FIG. 2



PROGRAMMABLE BREAKAWAY CLUTCH SYSTEM WITH COLLAPSIBLE FAILURE MODE

TECHNICAL FIELD

This invention relates to robotic tooling, and more particularly to an overload sensitive clutch for connecting a tool to the end of a robot arm.

BACKGROUND OF THE INVENTION

A problem in the use of robots in tooling operations is the problem of overload on the robot arm. This may occur if the robot arm or tool collides with something or if the tool gets stuck in the workpiece.

Several devices have been proposed and are in use that detect stresses and overloads on the robot arm and immediately shut down the robot when overload is detected in order to avoid damage to the robot or to the workpiece. These devices include expandable breakaway joints and connectors that break away when the robot arm experiences an overload.

For example, the devices shown in our U.S. Pat. Nos. 4,717,003 and 4,848,546 are expandable clutches useful in most robotic tooling situations. It has been found, however, that a collapsible clutch is preferable for use when the tool is an arc welding electrode, as collisions with the work piece are a more common occurrence. An expandable clutch as shown in our previous patents may not trip at a low enough threshold to protect the welding torch from damage if a collision puts the clutch into a compression failure mode. It has also been found that the ball and socket type of connection utilized in our expandable clutches is inappropriate for use in a collapsible clutch due to excessive friction between the parts, causing nonrepeatability.

SUMMARY OF THE INVENTION

The invention includes an adjustable clutch for use with robot end-of-arm tooling that provides a means for sensing movement of a tool attached to the clutch. The clutch includes a sealed piston movable member held within a chamber. The movable member is attached by a link to a tool connecting toggle. The piston is capable of lateral movement within the chamber, and the sensitivity of the breakaway clutch is adjusted by biasing the movement of the piston. In the preferred embodiment, the piston is biased using spring means and by introducing pressurized air into the piston chamber. A sensing device detects movement of the toggle relative to the clutch to shut down the robot when an overload is encountered.

The present invention provides a breakaway clutch having a collapsible failure mode. The clutch is ideal for use with arc welding tooling. The clutch has a sensor capable of detecting movement of the tool in relation to the clutch and of tripping an emergency stop when the movement is beyond a predetermined range of movement allowed. The clutch restores itself to a normal position when the overload is relieved.

According to one embodiment of the invention, the clutch includes a clutch housing that defines a piston cavity, a piston contained within the cavity, and a toggle that is attached to the piston through the housing by a link. The clutch includes biasing means to bias the piston in a direction toward the toggle. The toggle has locating means for positioning the piston in a normal operational position in relation to the housing and proximity sensing means for sensing deviations of the toggle

from its normal position. In a preferred embodiment, the link is a capsule-shaped solid body engaged with frustoconical/spherical internal walls in the piston and toggle.

BRIEF DESCRIPTION OF THE DRAWINGS

The foregoing and additional objects and advantages of the invention will be more apparent when the following detailed description is read in conjunction with the accompanying drawings, wherein like reference characters denote like parts in all views and wherein:

FIG. 1 is a side view of a robot arm utilizing the clutch of the present invention;

FIG. 2 is a sectional view of the clutch of FIG. 1 taken along lines 2—2;

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cated opening defined by wall 118. Front housing cover 114 and housing body 108 are fixed to rear mounting plate 102 by way of bolts 120 and locating pins 122.

A movable member includes piston 130 and O-ring 132 confined within the cavity formed by walls 111, 113 and 116. Wall 113 is cylindrical about central axis 109. Movable member 128 is capable of linear movement along central axis 109 from a normal position within the cavity shown in FIG. 3A wherein piston 130 is spaced apart from rear cavity wall 111. Movable member 128 is movable in the direction of rear cavity wall 111 to a position where piston 130 contacts wall 111, as shown in FIG. 3B. Movable member 128 includes an inwardly tapering frustoconical internal side wall 134 joined to a semi-spherical internal end wall 136. Internal side and end walls 134 and 136 are coaxial with central axis 109 and open in the direction of front cavity wall 116. A tool connecting member 140 includes toggle 22 and arm 28. Tool connecting member 140 has an external end 142 adapted for connecting clutch 10 to a tool. Tool connecting member 140 has an internal end 144 within the cavity, and internal end 144 has inwardly-tapering frustoconical internal first and second side walls 146 and 148, respectively, joined to a semi-spherical internal end wall 149. Internal side and end walls 146, 148 and 149 are coaxial with central axis 109, and open in the direction of the rear cavity wall 111.

Link 150 spans between tool connecting member 140 and movable member 128. Link 150 is a capsule-shaped solid body having a cylindrical side wall 151. Link 150 has semi-spherical external end walls 152 and 154 engaging with internal end walls 136 and 149, respectively, in the movable member and tool connecting member. Movable member 128 and tool connecting member 140 are spaced apart by link 150, and forces upon tool connecting member 140 are applied to movable member 128 by way of link 150.

Tool connecting member 140 is located within the cavity by way of conical seats 160 and spherical pins 162. Preferably, five seats 160 and pins 162 are provided in an asymmetrical pattern to provide a single return orientation for tool connecting member 140. Seats 160 are fixed in front housing cover 114. Tool connecting member 140, link 150 and movable member 128 are dimensioned such that movable member 128 is in the normal position shown in FIG. 3A when tool connecting member 140 is located such that pins 162 are seated in seats 160. Tool connecting member 140 is movable in all directions except in the direction of front cavity wall 116.

A spring 170 biases movable member 128 towards the normal position established by the linkage to tool connecting member 140 and the locating pins and seats. Movable member 128 is also biased towards the normal position by way of variable air pressures applied through air line 26 to the cavity by way of passageway 172. Thus, the sensitivity of clutch 10 to forces and moments applied to the movable member 128 by way of the tool connecting member 140 is variable.

Movement of the movable member 128 away from the normal position shown in FIG. 3A is sensed by way of a microswitch 180 fixed within electrical connector 24. Microswitch 180 is actuated by a bearing ball 182 constrained for linear movement within a cylindrical wall 184 formed in housing body 108.

Referring now to FIGS. 4 and 5, internal side wall 134 in piston 130 preferably has a taper A of about 13°. First internal side wall 146 of toggle 22 preferably has a

taper B of about 137°, and second internal side wall 148 preferably has a taper C of about 52°.

In operation, any compression force on tool connecting member 140 having a component in the Z direction or a moment having a component in the X or Y direction will apply force by way of link 150 to movable member 128. If the force and/or moments exceed a threshold predetermined by the bias on movable member 128 by way of spring 170 and air pressure in line 26, movable member 128 will shift away from the normal position shown in FIG. 3A to a position exemplified by FIG. 3B. In FIG. 3B, movable member

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forces upon said tool connecting member are applied to said movable member;
means for locating said tool connecting member such that when said movable member is in said normal position said tool connecting member is located with respect to said front cavity wall and movable in all directions except in the direction of said front cavity wall;
biasing means for biasing said movable member toward said normal position established by said means for locating, said biasing means being adjustable to vary the bias and thereby vary the sensitivity of said clutch to forces applied to said movable

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member by way of said tool connecting member; and
sensing means for sensing movement of said movable member away from said normal position.
2. The clutch of claim 1 where said link is a solid body having a cylindrical side wall between said semi-spherical end walls of said link.
3. The clutch of claim 1 wherein said frustroconical internal side wall in said movable member has a taper of about 13 degrees.
4. The clutch of claim 1 wherein said frustroconical internal side wall in said tool connecting member has a taper of about 52 degrees.

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